

Lecture 16

①

Stream functions

Mass cons. of irrotational flows

$$\vec{\nabla} \cdot \vec{u} = 0$$

$$\frac{\partial u_i}{\partial x_i} = 0$$

In 2-D plane flow:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad \text{①}$$

↳ now define $\psi(x, y, t)$ s.t.

Sub
this
into
streamline
eq.

$$\Rightarrow \boxed{u = \frac{\partial \psi}{\partial y}, \quad v = -\frac{\partial \psi}{\partial x}}$$

Sub. this into ①

$$\frac{\partial}{\partial x} \left(\frac{\partial \psi}{\partial y} \right) + \frac{\partial}{\partial y} \left(-\frac{\partial \psi}{\partial x} \right) = \frac{\partial^2 \psi}{\partial x \partial y} - \frac{\partial^2 \psi}{\partial y \partial x} = \underline{\underline{0}}$$

Physical
+ Interpretation:

recall Streamlines:

$$\frac{dx}{u} = \frac{dy}{v}$$

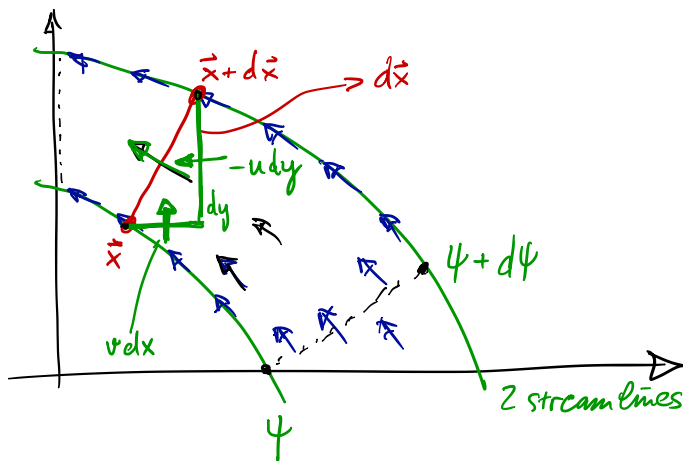


$$\underline{v dx - u dy = 0}$$

$$d\psi = \boxed{\frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy = 0}$$

∴ $d\psi = 0$ along streamlines
i.e. streamlines are contours of the stream fn.

(2)



Consider the
volume flux
between two
streamlines

$$v dx + (-u) dy = -\frac{\partial \psi}{\partial x} dx - \frac{\partial \psi}{\partial y} dy = -d\psi$$

→ volume flow rate between two
streamlines is just the difference in their
 ψ values.

In vector notation:

$$\hat{k} = (0, 0, 1)$$

$$\vec{u} = \hat{k} \wedge \vec{\nabla} \psi$$

$$u_i = \epsilon_{ijk} \delta_{j3} \frac{\partial \psi}{\partial x_k} = -\epsilon_{ikj} \delta_{j3} \frac{\partial \psi}{\partial x_k} = -\epsilon_{ik3} \frac{\partial \psi}{\partial x_k} =$$

$$u_i = \epsilon_{i3k} \frac{\partial \psi}{\partial x_k} = \epsilon_{i31} \frac{\partial \psi}{\partial x_1} + \epsilon_{i32} \frac{\partial \psi}{\partial x_2}$$

$$u_1 = -\frac{\partial \psi}{\partial x_2}, \quad u_2 = \frac{\partial \psi}{\partial x_1}$$

e.g.

$$\vec{u} = \vec{a} \wedge \vec{b}$$

$$u_i = \epsilon_{ijk} a_j b_k$$

Conservation of Momentum

(3)

Newton's 2nd Law

↳ for a material volume $V(t)$
with an area $A(t)$

ρu - m/m per unit volume

$$\frac{d}{dt} \int_{V(t)} \rho(\vec{x}, t) \vec{u}(\vec{x}, t) dV = \int_{V(t)} \rho(\vec{x}, t) \vec{g} dV + \int_{A(t)} f(\hat{n}, \vec{x}, t) dA$$

$\sim ma$

$F = F_{\text{body}} + F_{\text{surface}}$

↓ use Reynold's
Trans. Thm:

$$\int_V \frac{\partial}{\partial t} (\rho \vec{u}) dV + \int_A \rho \vec{u} (\vec{u} \cdot \hat{n}) dA = \int_V \rho \vec{g} dV + \int_A f(\hat{n}) dA$$

integral form of m/m balance (for material volume)

⇓

mtm equiv.
to
mass cons.:

$$\left[\int_V \frac{\partial \rho}{\partial t} dV + \int_A \rho (\vec{u} \cdot \hat{n}) dA = 0 \right]$$

⇓

turn into the
differential form

→ see Kundu 4.4.

Cauchy's Eq. of Motion

(4)

$$\rho \frac{D u_i}{D t} = \rho g_i - \frac{\partial}{\partial x_j} T_{ij}$$

↓ stress tensor

$$f_i = n_j T_{ij}$$

3 eqns

+ 1 eqn from mass cons. (cont. equation)

+ 2 eqn from thermodynamics

6 eqns.

Unknowns :

$$\left. \begin{array}{l} u_j - 3 \text{ unknowns} \\ \rho - 1 \text{ unknown} \\ \underline{T_{ij}} - 9 \text{ unknowns} \end{array} \right\} 13 \text{ unknowns}$$

need to
relate stresses to deformation (strain)
rates

$$\downarrow \quad \left(\frac{\partial u_i}{\partial x_j} \leftrightarrow T_{ij} \right)$$

Constitutive eqns